

(3) A loading in the longitudinal direction equal to the weight of the tank and the heaviest product carried.

(g) *Owner* means the person, corporation, company, partnership, or organization in which is vested the ownership, dominion, or title of a portable tank.

[CGD 73-172, 39 FR 22950, June 25, 1974, as amended by CGD 84-043, 55 FR 37409, Sept. 11, 1990]

#### § 64.9 Maintenance, repair, and alteration of MPTs.

(a) Each MPT must be maintained in accordance with the approved plans, this part, and subpart 98.30 of this chapter.

(b) Repair of an MPT is authorized, provided that each repair is in accordance with the approved plans.

(c) No MPT may be altered, except with the written approval of the Commanding Officer, U.S. Coast Guard Marine Safety Center.

(d) After each welded repair or alteration, an MPT must be hydrostatically pressure-tested in accordance with paragraph (a) of § 64.83 of this part.

[CGD 84-043, 55 FR 37409, Sept. 11, 1990]

### Subpart B—Standards for an MPT

#### § 64.11 Design of MPTs.

An MPT must be designed—

(a) In accordance with the ASME Code and this subpart;

(b) With a maximum gross weight of 55,000 pounds;

(c) To hold a liquid cargo that has a vapor pressure of 43 pounds per square inch absolute (psia) or less at a temperature of 122 °F;

(d) With a minimum service temperature of 0 °F or higher;

(e) With a maximum allowable working pressure of not less than 20 pounds per square inch gauge (psig) but not more than 48 psig; and

(f) To withstand dynamic loading conditions applied simultaneously.

[CGD 84-043, 55 FR 37410, Sept. 11, 1990; 55 FR 40755, Oct. 4, 1990]

#### § 64.13 Allowable stress; tank.

(a) The calculated stress in the tank under design conditions, including dynamic loading conditions applied si-

multaneously, must not exceed the allowable stress listed in Division 1 of section VIII of the ASME Code, for a design temperature of 122 °F.

(b) The calculated stress in the tank at test pressure must not exceed 75 percent of the minimum yield stress,<sup>1</sup> or 37.5 percent of the minimum tensile stress<sup>1</sup> of the material, whichever is less.

[CGD 73-172, 39 FR 22950, June 25, 1974, as amended by CGD 84-043, 55 FR 37410, Sept. 11, 1990]

#### § 64.15 Allowable stress; framework.

The calculated stress for the framework must be 80 percent or less of the minimum yield stress of the framework material under the dynamic loading conditions that are applied simultaneously.

#### § 64.17 Minimum tank thickness.

(a) Except as allowed in paragraph (b) of this section, a tank with a diameter of—

(1) 6 feet or less must have a shell and head of  $\frac{3}{16}$  inch thickness or more; or

(2) More than 6 feet must have a shell and head of  $\frac{1}{4}$  inch thickness or more.

(b) If the tank has additional framework to guard against accidental puncturing of the tank, the shell and head thickness must be  $\frac{1}{8}$  inch or more.

#### § 64.19 External pressure.

(a) A tank without a vacuum breaker must be designed to withstand an external pressure of  $7\frac{1}{2}$  psig or more.

(b) A tank with a vacuum breaker must be designed to withstand an external pressure of 3 psig or more.

#### § 64.21 Material.

The material for a tank must meet the requirements in Division 1 of section VIII of the ASME Code.

[CGD 73-172, 39 FR 22950, June 25, 1974, as amended by CGD 84-043, 55 FR 37410, Sept. 11, 1990]

#### § 64.23 Gasket and lining.

Each gasket and lining must be made of material that is—

<sup>1</sup>Listed in Division 1 of section VIII of the ASME Code.